

| <b>P13CV33</b> Page No 2 |  |   |
|--------------------------|--|---|
| c.                       | At a certain point in piece of elastic material there are normal tensile stresses $f_x = 120$ MPa,     |   |
|                          | $f_y = 60$ MPa. In addition there is a positive shearing stress (Left up right down) $q = 80$ MPa.     |   |
|                          | Determine;   | 0 |
|                          | i) The magnitude and directions of the principal stresses  | 8 |
|                          | ii) The magnitude and direction of the maximum shearing stress   |   |
|                          | iii) The normal and shearing stress on a plane inclined at 30° to the direction of 120 MPa.            |   |
| 4 a.                     | Derive an expression for circumferential stress $(f_1)$ and longitudinal stress $(f_2)$ in the case of | 4 |
|                          | thin cylinder.   | 4 |

b. Derive expressions for change in diameter, change in length and change in volume of a thin cylinder.

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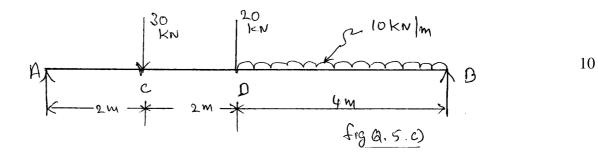
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c. A thick cylinder of 400 mm material diameter and 100 mm thickness contains a fluid at a pressure of 80 N/mm<sup>2</sup>. Find the maximum and minimum hoop stresses across the section. 8 Also sketch the radial and hoop stress distribution across the section.

## **UNIT - III**

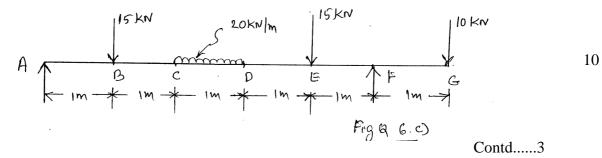
- Define bending moment, shear force, BMD and SFD. 5 a.
  - Derive relationship between intensity of uniformly distributed load w, shear force F and b. 6 bending moment M.
  - c. Draw BMD and SFD for the beam shown in Fig. Q. 5(c).



- 6 a. Define point of contra flexure, bending moment (Hogging and sagging bending moment) and shear force (positive and negative shear force).
  - b. For a simply supported beam carrying uniformly distributed load  $\omega/m$  over entire span l,

Show that maximum B.M,  $M_{\text{max}} = \frac{\omega l^2}{8}$  at  $x = \frac{l}{2}$  from either support.

c. Draw BMD and SFD for the beam shown in Fig. Q 6(c). Find the point of contra flexure and maximum bending moment.



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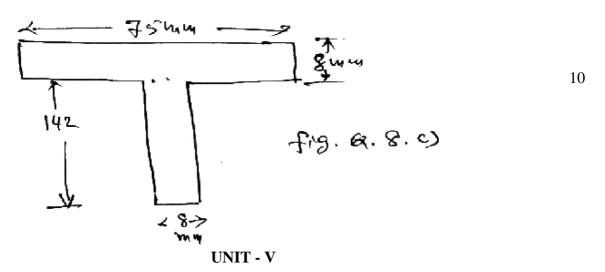
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## UNIT - IV

- 7 a. State assumptions mode while deriving (Bernauli) bending equation.
  - b. Derive bending equation (Bernauli equation) with usual notations.
  - c. A wooden beam 10 m long 360 mm deep and 300 mm wide is simply supported and loaded with a uniformly distributed load. Find the safe total load. Factor of safety = 6, maximum 10 shear intensity of the material is = 60 MPa.
- 8 a. Draw shear stress variation diagram for various standard sections (including triangle, rhombus and T - Section).
  - b. Show that maximum shear stress is 1.5 times average shear stress for a rectangular section.
  - c. A T-Section beam shown in Fig. Q 8(c) is subjected to a shear force of 9 kN at a section.
    Determine the amount of maximum intensity of shear stress and draw the distribution of shear stress across the depth of the section.



- 9 a. State assumptions made while deriving deflection equation.
  - b. Derive deflection equation with usual notations.
  - c. A simply supported beam has a span of 6 m. It carries two concentrated loads of 40 kN and 10 kN at distance of 2 m and 4 m from left hand support. Find the deflection under the 40 kN load and the position and magnitude of maximum deflection. E = 200 GPa and  $I = 40 \times 10^{-6} \text{ m}^4$ .
- 10 a. Derive torque equation with usual notations.
  - b. State assumptions made while deriving torque equation.
  - c. Calculate the dimensions of a hollow steel shaft to transmit 600 kW at a speed of 120 r.p.m. the maximum torque being 1.12 times the mean (12% extra). The internal diameter of the shaft is 60% of the outside diameter and the greatest intensity of shear stress in the steel is limited to 28 MN/m<sup>2</sup>.